# The solid medium deformation apparatus — reloaded

Renée Heilbronner  $(1)^{(2)}$ , Holger Stunitz  $(2)^{(3)}$ , Jacques Précigout (3), and Hugues Raimbourg (3)

- (1) Geological Institute, Basel University, Switzerland, (renee.heilbronner@unibas.ch)
- (2) The Arctic University, Tromsø, Norway
- (3) Institut des Sciences de la Terre d'Orléans

#### EGU2020-9859 - Abstract

Rock deformation experiments are used to compile mechanical data sets for minerals and rocks and to study microstructure and texture development.

The Griggs apparatus, a solid medium piston cylinder machine was designed about 60 years ago to investigate rock deformation mechanisms and rheology at elevated confining pressures. In a typical experiment today, the confining medium is NaCl, with confining pressures up to 3 GPa, temperatures up to 1100°C, and displacement rates between 10-8 and 10-2 ms-1 (equivalent to strain rates of 10-7 to 10-3 s-1). In axial tests, the cylindrical samples are 12 to 15 mm long with a diameter of 0.625 mm. In shearing test, split cylinder assemblies are used with 0.5 to 1 mm thick samples introduced along the 45° pre-cut. Reasonable total strains are limited to 30% axial shortening or shear strains of gamma 4. (Higher strains can be attained but are difficult to analyse mechanically. Unlike for gas rigs, torsion is not available for solid medium machines).

As of now, the operational fleet of solid medium deformation apparatus comprises worldwide over 20 machines in different labs (mainly in Europe, U.S.A. and Japan), providing the scientific community with an evergrowing rheological and microstructural data base.



Participants of the Orléans Workshop on Experimental Solid Medium Rock Deformation, January 30-31, 2020

In view of numerous developments in experimental design, as well as improvements of hardware and software for data acquisition and processing, the experimental community was recently invited to a two-day workshop, hosted by the experimental group of Orléans University.



The main goal was to discuss the following points:

- how to further improve the apparatus, increase its scope and improve calibrations;
- how to further improve data processing, and the precision and reliability of the results;
- how to maintain consistency among the labs and through time (backwards compatibility); - how ensure compatibility of results from axial and shearing experiments;
- how to make the data available to the community.

### EGU2020-9859 - Display

This display focuses on the software used for converting the recorded experimental data to stress-strain curves. In particular, on the choices that have to be made on the way and how they influence the results. It is proposed to make every step transparent such hat different labs publish coherent results.



## converting experimental data to stress-strain curves ...

... in a transparent fashion

```
general procedure
```

go to software - rigP(prepare), rigC(for axial) and rigS(for shear)

pre-process – create input file

go to prepare raw data from experimental record

## run program with explicit options:

option A go to crop data for analysis – select hitpoint

option B go to perform 'friction' correction ... if you must

option C go to 'area corrections' – for axial and shear experiments

option D  $g_0 t_0$   $\sigma_3$  during the experiment – the salt correction

option E  $g_0 t_0$   $\sigma_1$  and  $\sigma_3$  at the start of the experiment

summary of options – corrections and calculations



## software - rigP(prepare), rigC(for axial) and rigS(for shear)

#### prepare input

### rigP

#### necessary input:

- run record (machine data)
- metadata apparatus experimental conditions sample geometry sample assembly

#### output:

- raw data file of complete run (SI units)
- input file for rigC and rigS: reduced file length (max = 1000 pts) (smoothing of data not yet implemented) includes both hitpoints (classical and 'lead')

## explicit options

```
*--- A-select hitpoint
        write(6,'(a)') 'Select hitpoint (1=classical, 2=new(=lead)) '-
*---B-select · friction · correction-
        ·write(6,'(a)') · 'Friction · correction · for · F · ? · (1=yes · · 0=no) ' ¬
        read(5,*) ioptionFRIC-
*---·C-choice·of·area·correction·(Poisson·correction)
        ·write(6,'(a)') · 'Options · for · area · correction'¬
        ·write(6, '(a)') · '0: ·No · area · correction' ¬
        ·write(6,'(a)') · '1: ·Homogeneous · shortening · of · sample · '¬
        ·write(6,'(a)') · '2: ·Barreling · of · sample' ¬
        read(5,*) · ioptionAREA¬
*---·D-definition·of·sig3¬
        ·write(6,'(a)') · 'Definition · of · sig3(t)'-
        ·write(6, '(a)') · '1: ·sig3(t) ·= ·Pc(0) ·at ·start'¬
        ·write(6,'(a)') · '2: ·sig3(t) ·= ·Pc(0) ·+ ·SALT · correction'¬
        \cdotwrite(6, '(a)') \cdot '3: \cdotsig3(t) \cdot= \cdotPc(t) \cdotas \cdotmeasured'
        read(5,*) ioptionSIG3-
       --salt-correction-is-only-possible-for-ioptionSIG3=2-
        if(ioptionSIG3.eq.2) ioptionSALT=1-
*---·E-definition·of·sig1·and·sig3·at·start·of·experiment·(time=0)¬
        write(6,'(a)') 'Defining sig1(0) and sig3(0) at time=0'-
        ·write(6,'(a)')·'1:·sig1(0)=sig3(0)=pc(0)'¬
        write(6,'(a)') '2: sig1(0) = sig3(0) = 1/16*F(0)/A(0) + 15/16*pc(0)'-
        write(6,'(a)') · '3: · sig1(0) = F(0) / A(0) · and · sig3(0) = pc(0) ' ¬
        read(5,*) ioptionSTART¬
```

#### analyze data

#### rigC - for axial experiments

- open input file
- asks for options
  - A choice of hitpoint
  - B 'friction correction' −Y/N
  - C area correction
  - D confining pressure correction
  - E set starting values for  $\sigma_1$  and  $\sigma_3$
- data is read from hitpoint to end (option A)
- stiffness correction of  $d \rightarrow dc$
- strains and strain rates are calculated
- friction' correction of  $F \rightarrow Fc$  (option B)
- calculate cross sectional area (option C)
- define  $\sigma_3$  and slope ( $\Delta MPa/mm$ ) (option D)
- calculate  $\Delta \sigma = (Fc Fc(0)) / area$
- determine  $\sigma_1(0)$  and  $\sigma_3(0)$  at start (option E)
- 11. derive  $\sigma_1 = \sigma_3(0) + \Delta \sigma$
- calculate mean stresses
- calculate equivalent viscosity
- create output file

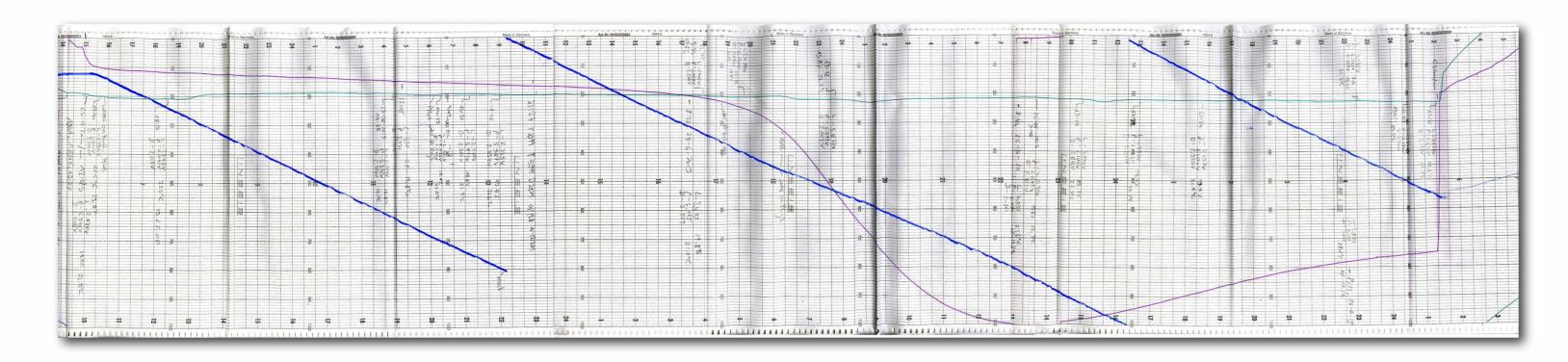
#### rigS — for shear experiments

- open input file
- asks for options
  - A choice of hitpoint
  - B 'friction correction' Y/N
  - C area correction
  - D confining pressure correction
  - E set starting values for  $\sigma_1$  and  $\sigma_3$
- data is read from hitpoint to end (option A)
- stiffness correction of  $d \rightarrow dc$
- shear strains, shear strain rates are calculated
- friction' correction of  $F \rightarrow Fc$  (option B)
- calculate overlap area (option C)
- define  $\sigma_3$  and slope ( $\Delta MPa/mm$ ) (option D)
- calculate  $\Delta \sigma = (Fc Fc(0)) / area$
- determine  $\sigma_1(0)$  and  $\sigma_3(0)$  at start (option E)
- 11. derive  $\sigma_1 = \sigma_3(0) + \Delta \sigma$  (inside shear zone)
- calculate mean stresses
- calculate  $\tau$  and  $\sigma_n$  (inside shear zone)
- calculate equivalent viscosity
- create output file

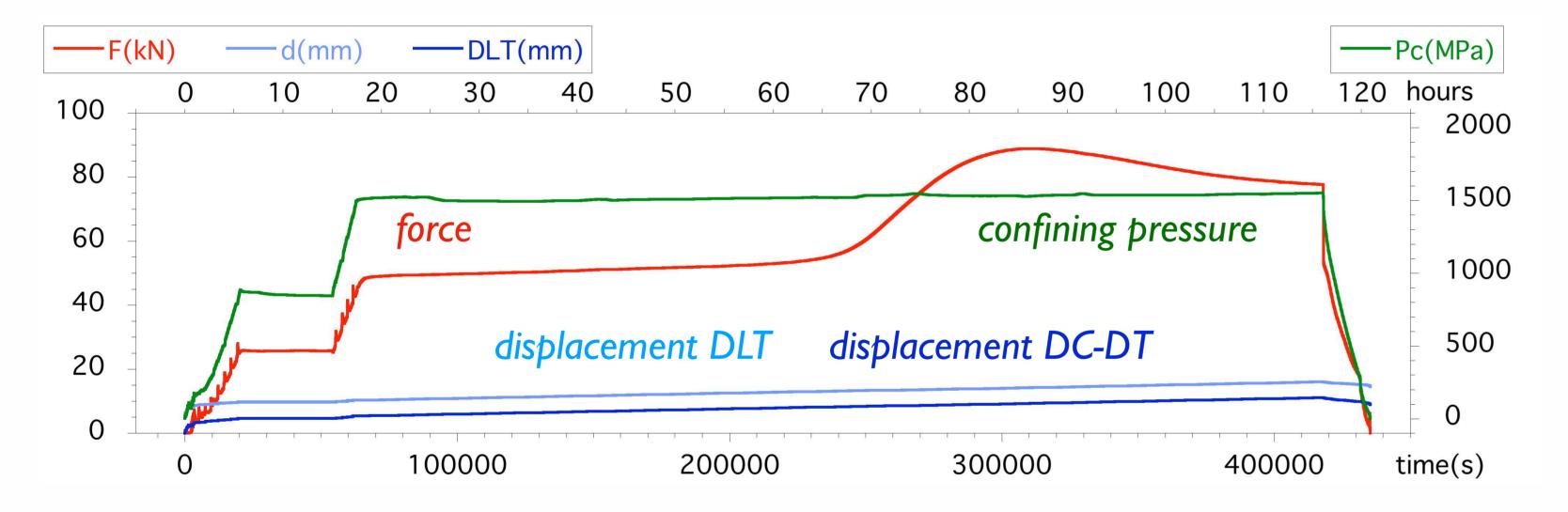
Fortran sources are available from <a href="https://micro.earth.unibas.ch/">https://micro.earth.unibas.ch/</a> or at <a href="renee.heilbronner@unibas.ch/">renee.heilbronner@unibas.ch/</a>

## prepare raw data from experimental record

#### run record



#### raw data = run record converted to SI units



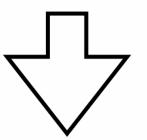
#### why it is important

Open data is generally agreed to be beneficial for science and scientists.

- a) published experimental data can be re-anaylized and compared to new data, in a coherent fashion, i.e. using the same options
- b) through the citation, the experimentalist is honoured if his or her data is re-evaluated.

#### meta data

apparatus experiment	distortion, run-in slope ('friction'), Pc, T, displacement rate,
assembly sample	confining medium, piston diameter, axial: length, diameter
•	shear: initial / final thickness, angle of pre-cut,



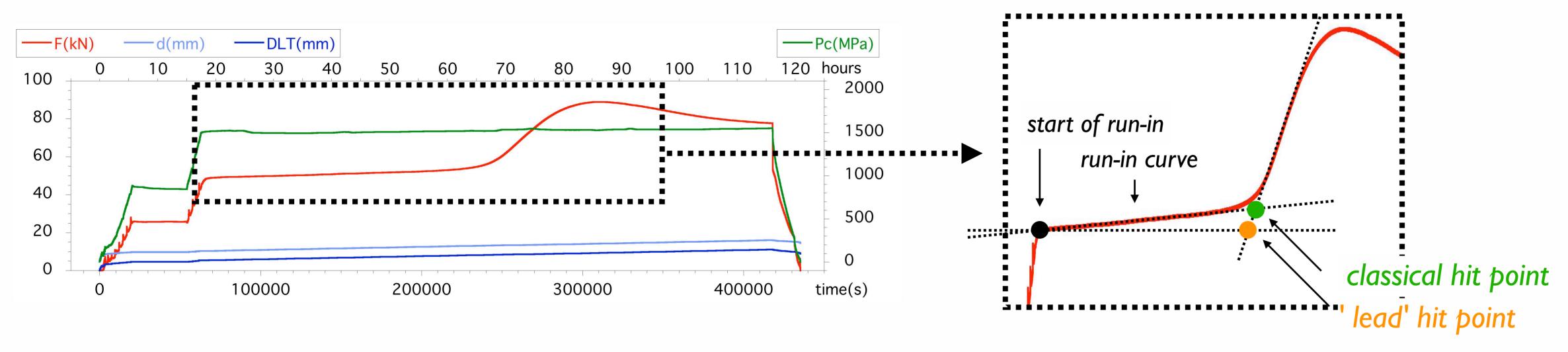
#### meta data (as used in header of input file for rigS2020)

name of experiment	383BR	
apparatus	Tromsø 2	
distortion(mm/N)	0.80000E-05	
friction(N/mm)	0.13000E+04	
nominal Pc(MPa)	1500	
nominal T(°C)	700	
displacement rate(ms-1)	10-8	
inner outer sleeve (1=NaCl 2=KI)	11	(all NaCl)
diameter(mm)	6.33	
length(mm)	0	(NA for shear)
alfa(deg)	45	
thO(mm)	0.90	
thFinal(mm)	0.58	
pre-experimental slip(mm)	0.35	
gamma meas	0	<pre>(not measured)</pre>



## crop data for analysis – select hitpoint

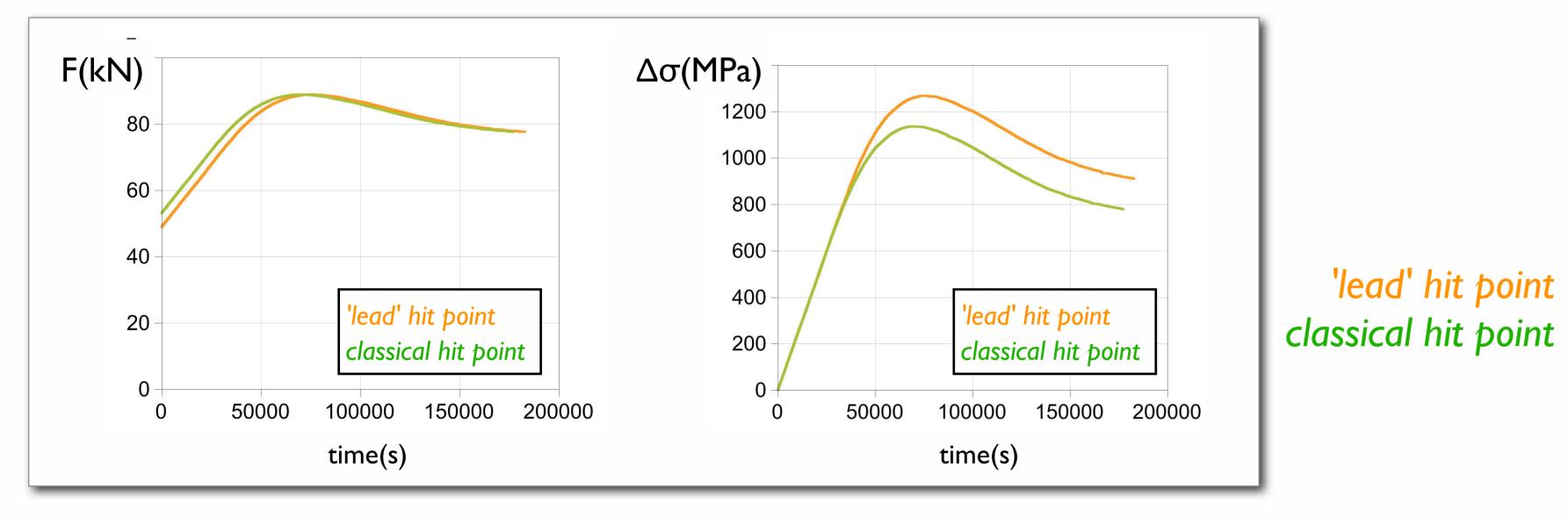
#### raw data file



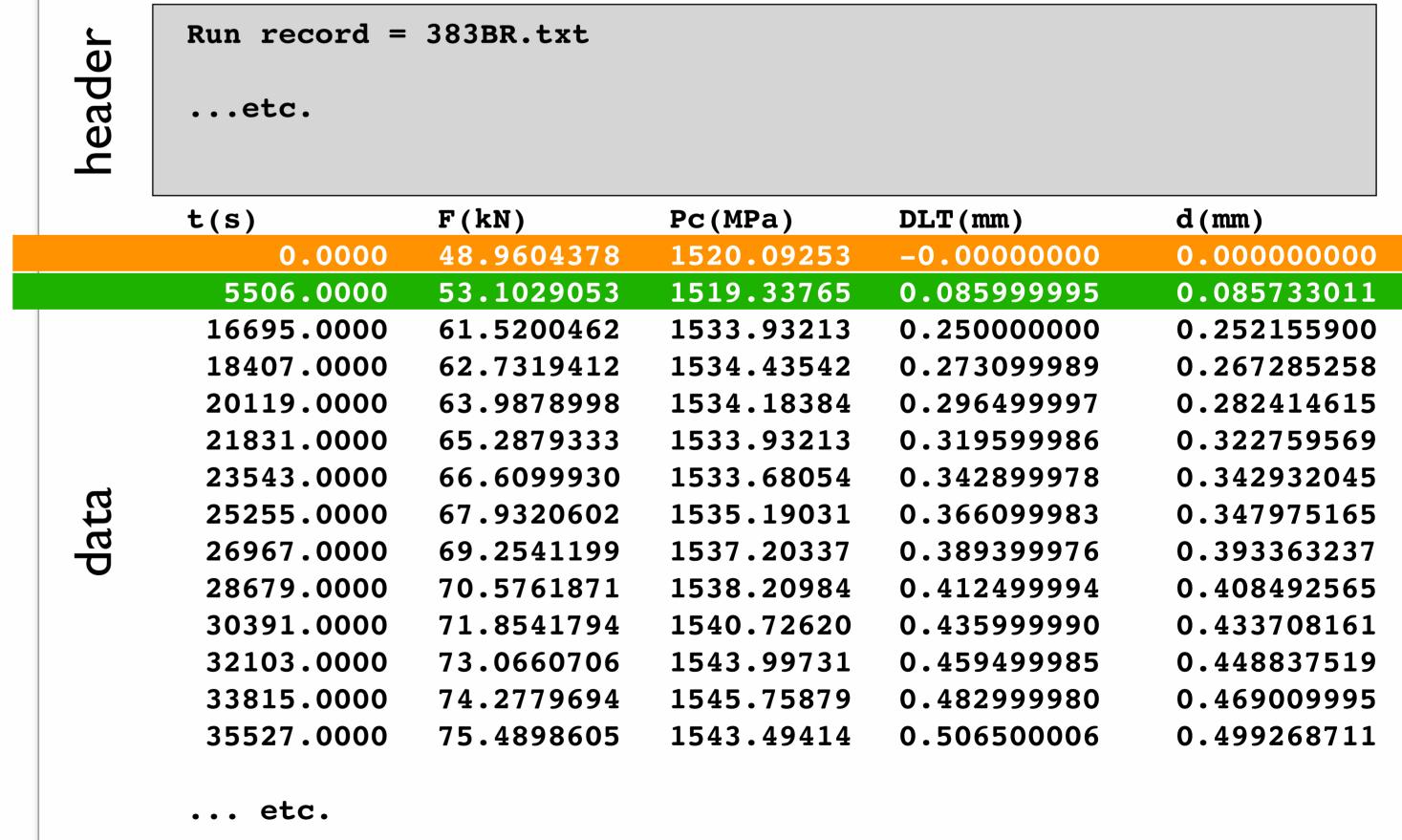
#### options A

- I: using classical hitpoint
- 2: using (new) lead hitpoint

#### effect of choice



input file



#### why it is important

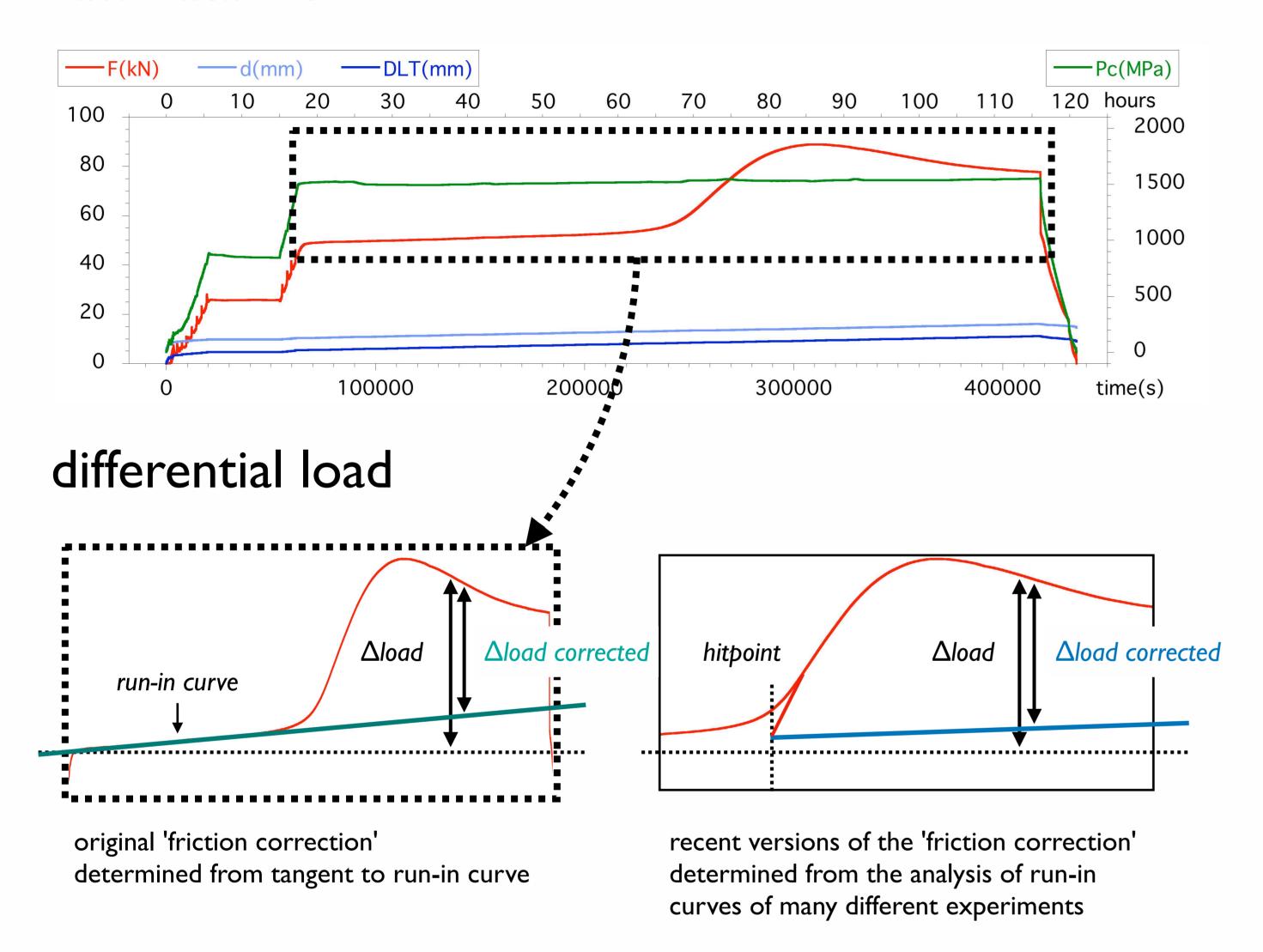
Selection decides if the load accumulated between the start and the end of the run-in curve is 'felt' by the sample as differential stress, i.e., if the load at hitpoint = load at 'lead hit', or if the load at hitpoint = load extrapolated from the run-in curve.



'lead' hit point

## perform 'friction' correction ... if you must

#### raw data file

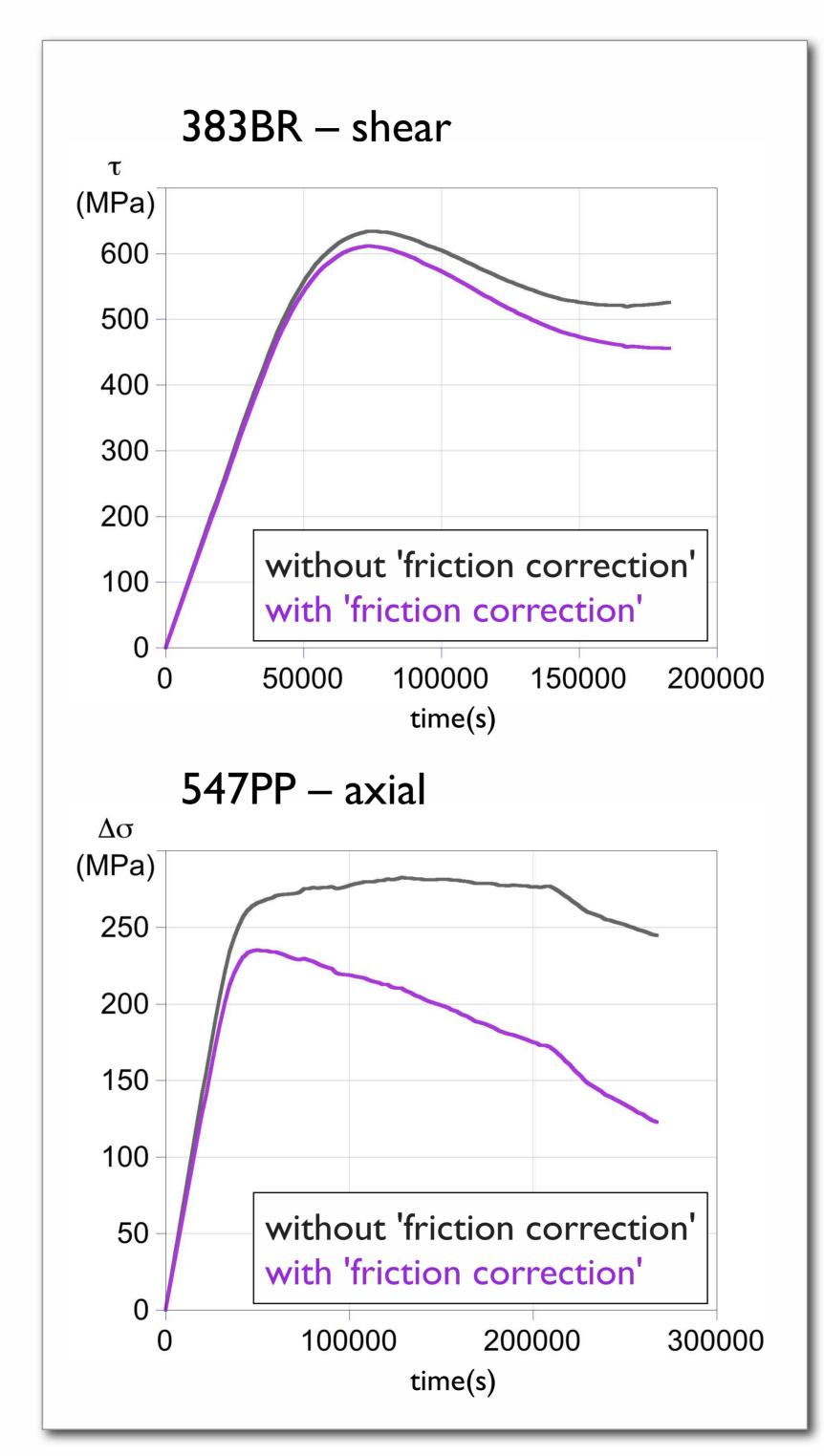


## why is there a 'friction correction'

The 'friction correction' is still used occasionally to curb apparent strengthening of sample at high strains.

However, there is only an empirical basis for this correction – it certainly has nothing to do with friction – and in cases where the differential load is small (weak samples), the resulting load may even become negative...

#### effect of 'friction correction'



#### options B

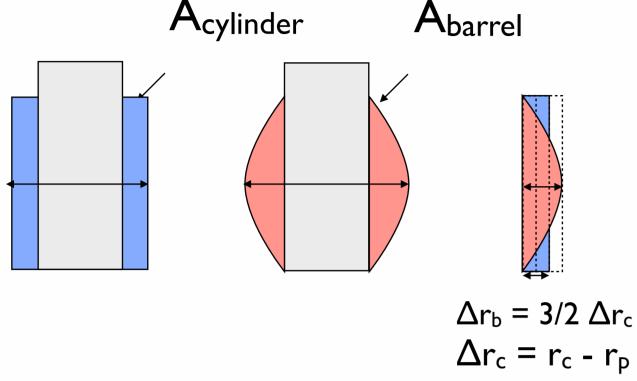
0: no 'friction? correction is applied I: using 'friction? correction

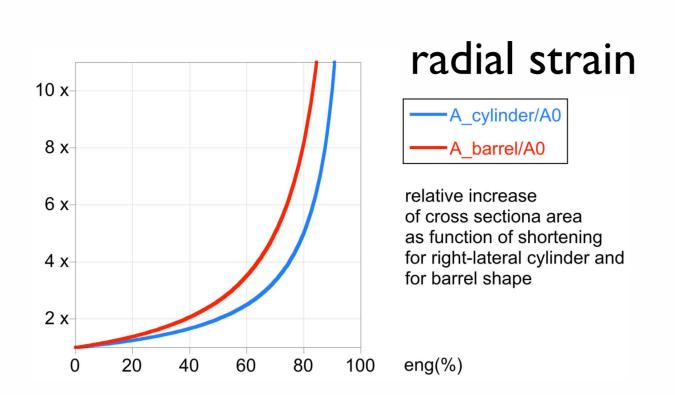
Compression experiments are started by moving the  $\sigma_I$ -piston in order to bring it into contact with the sample (hitpoint). During the run-in, the piston moves through lead and the load increases as a function of displacement. One can think of the slope of the run-curve as the base line with respect to which the differential load has to be calculated. Because the slope was originally attributed to friction between the  $\sigma_1$ - and the  $\sigma_3$ piston, this correction was called 'friction correction'.



## 'area corrections' – for axial and shear experiments

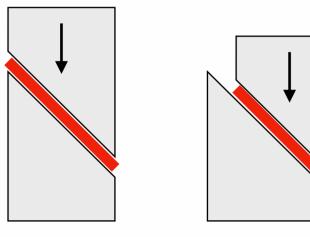
#### axial

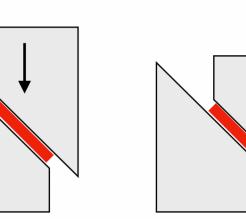


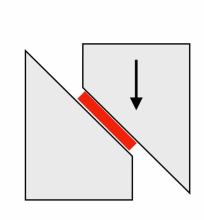


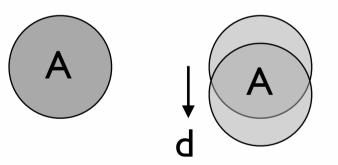
#### shear

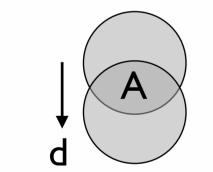
 $A = A_{\text{overlap}}$ 

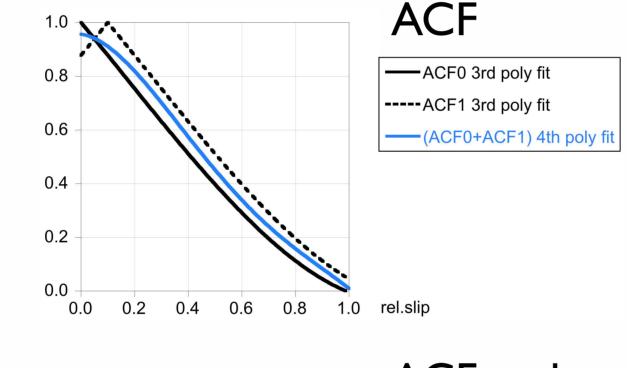


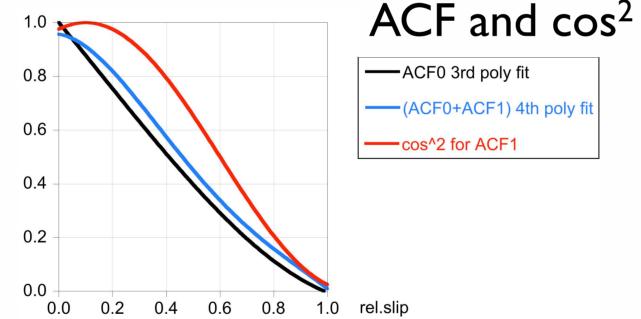








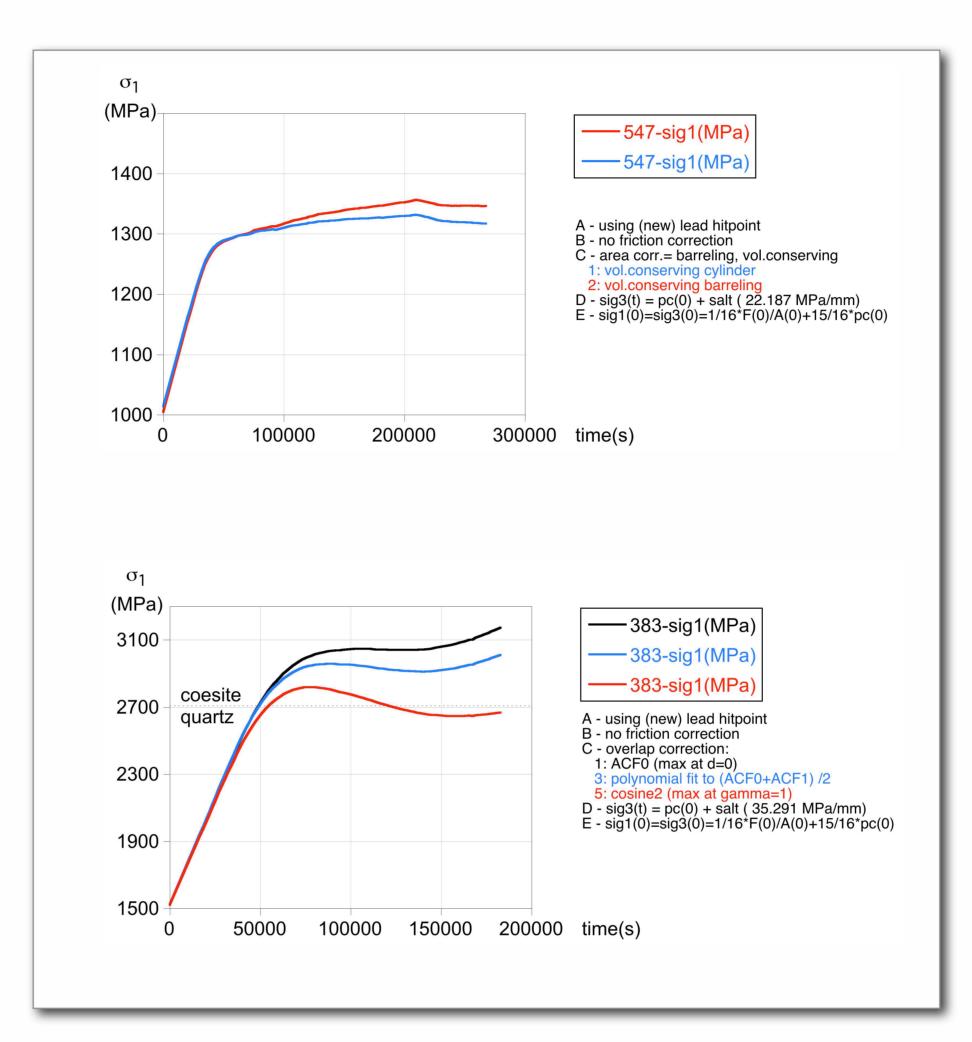


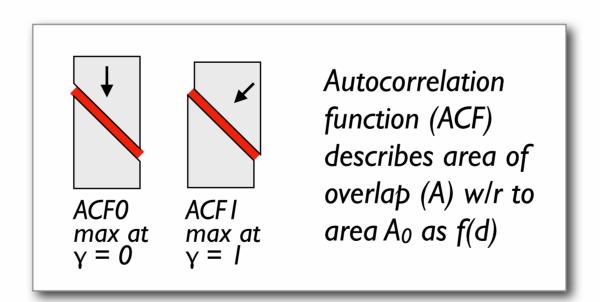


### why is it important

Both area corrections have a strong influence on the stresses, not only w/r to their absolute values, but also w/r to the general behaviour, i.e., wether a sample displays weakening, strengthening or steady state flow.

#### effect of area corrections





### options C- axial

- 0: no area correction
- I: homogeneous shortening of sample
- 2: barreling of sample

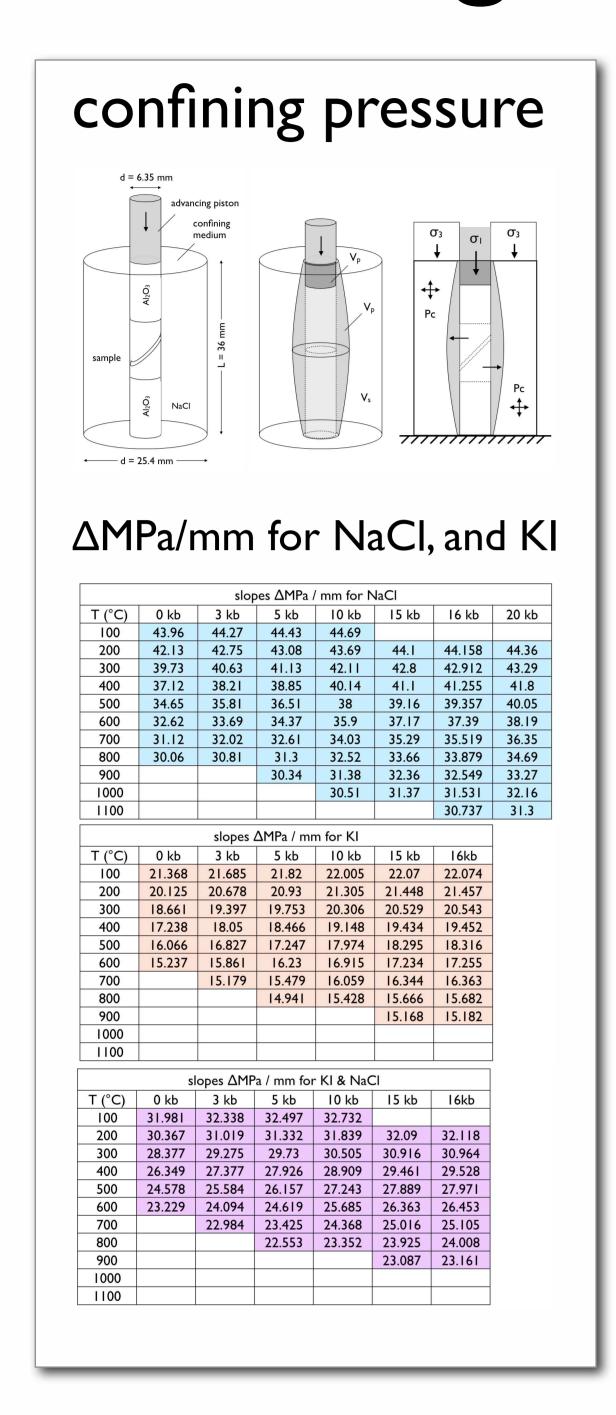
### options C – shear

- 0: no area correction
- I:ACF0 (max at d=0)
- 2:ACFI (max at gamma=I)
- 3: polynomial fit to (ACF0+ACFI) /2
- 4: cosine2 (max at d=0)
- 5: cosine2 (max at gamma=1)
- 6: cosine (max at d=0)

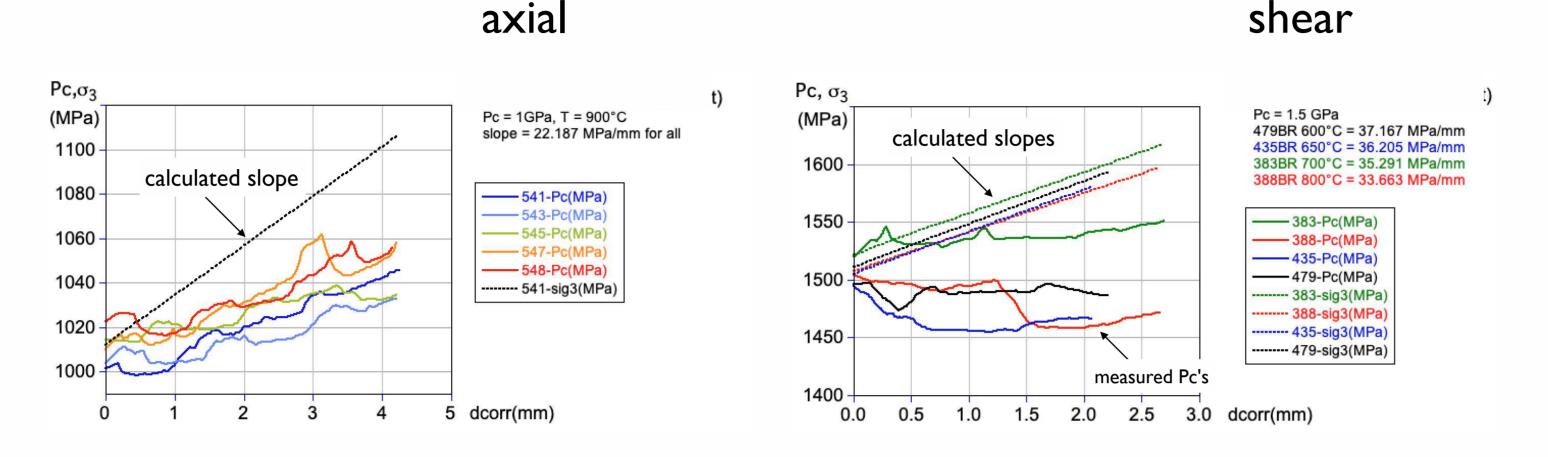
The area correction targets two different 'areas'. In axial experiments it is the cross sectional area of the sample which grows as the sample is shortened. In the case of shear experiments, the area to be corrected is the area of overlap of the forcing blocks which decreases as the forcing blocks are offset with increasing shear. Stresses in shear samples are notoriously difficult to assess. Sample 383BR (Richter et al., JGR, 2016 - which underwent both a qtz-to-coe and a coeto-qtz transition) is used to evaluate the different options for the overlap correction.



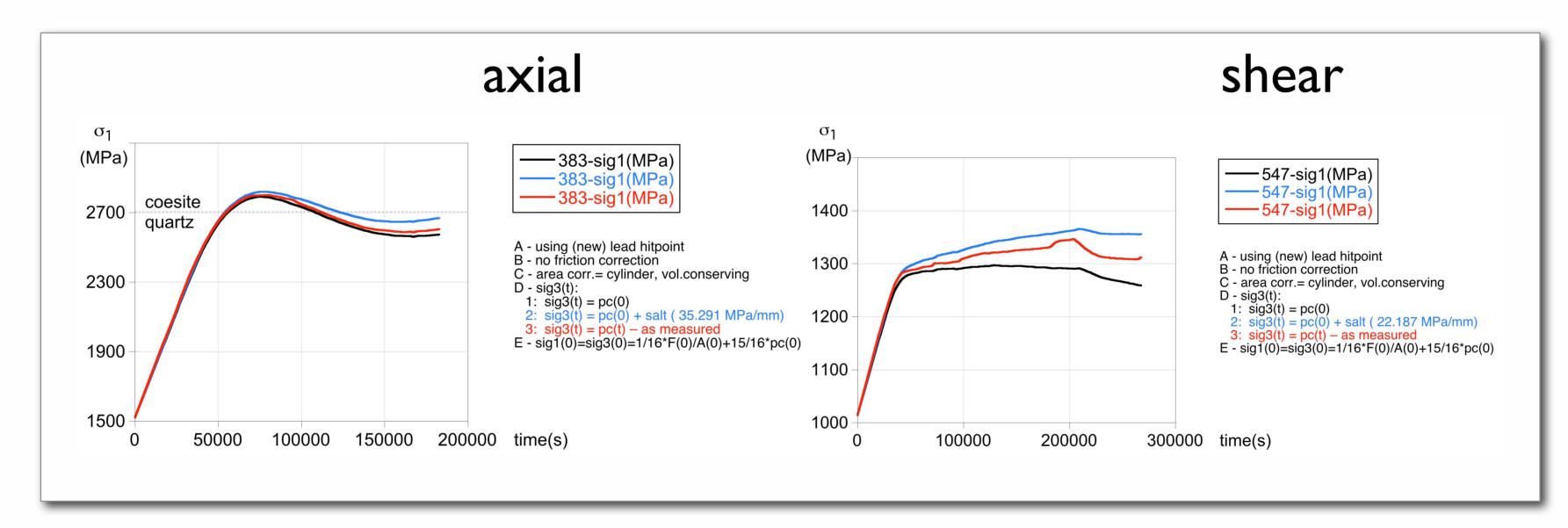
## σ<sub>3</sub> during the experiment – the salt correction



### measured and calculated confining pressure



#### effect of salt correction



### options D

I: sig3(t) = Pc(0) at start

2: sig3(t) = Pc(0) + SALT correction

3: sig3(t) = Pc(t) as measured

Traditionally the confining pressure was not measured during the experiment. Therefore  $\sigma_3(t)$  was set to the value of the confining pressure at the start.

Today we have the option of monitoring pc(t), however, these measurements are not all too reliable because we cannot measure the confing pressure directly. What is measured is the oil pressure of the hydraulic ram and it is not clear if this pressure is fully transmitted to the confining medium.

In addition it is to be expected that the pressure inside the vessel increases as the loading piston advances, thus introducing additional material into the fixed volume of the pressure vessel. This pressure increase only stops once the  $\sigma_3$ -piston starts to retreat.

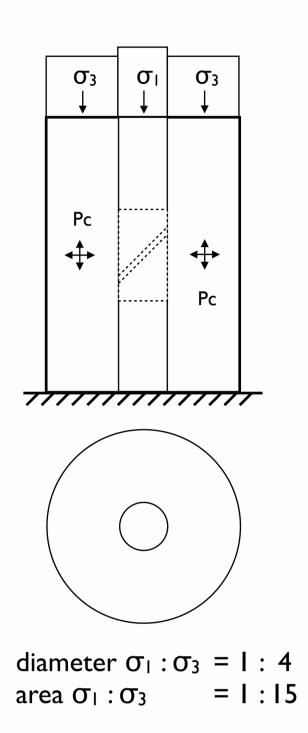
### why it is important

 $\sigma_1(t)$  cannot be calculated directly, but is found as the sum of the confining pressure and the differential stress:  $\sigma_1(t) = \sigma_3(t) + \Delta\sigma(t)$ .



## $\sigma_1$ and $\sigma_3$ at the start of the experiment

#### confining pressure



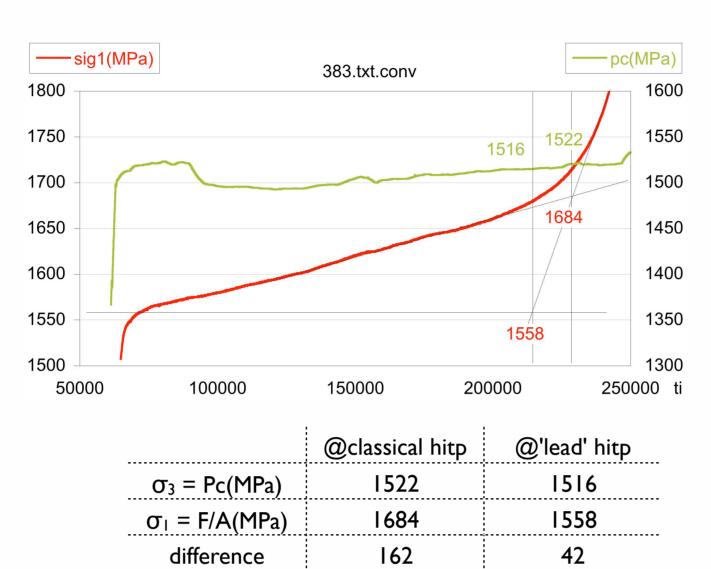
At the start of the experiment, the confing pressure is applied through the ring-shaped  $\sigma_{l}$ - piston and the so-called  $\sigma_{l}$ - or load piston. In general,  $\sigma_{l}$  (= load/(area of  $\sigma_{l}$ - piston)) is not the same as  $\sigma_3$  (= oil pressure of hydraulic ram that actives the  $\sigma_3$ - psiton.

#### why it is important

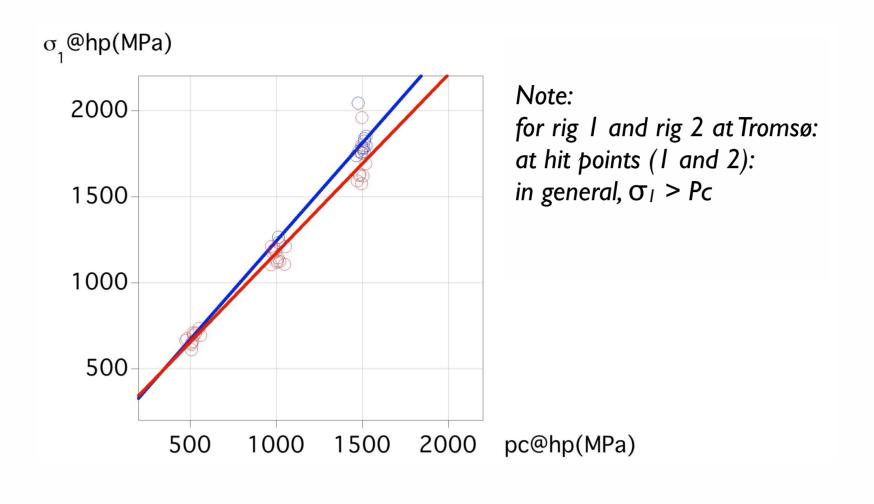
In the context of phase transformations, for example, it may be critical to know the absolute stress levels of  $\sigma_I$  or  $\sigma_{mean}$ . And because absolute stress levels depend on the starting values,  $\sigma_1(0)$ and  $\sigma_3(0)$  we should determine these values as correctly as possible.

### measured s1 and s3 at start of experiment

1519

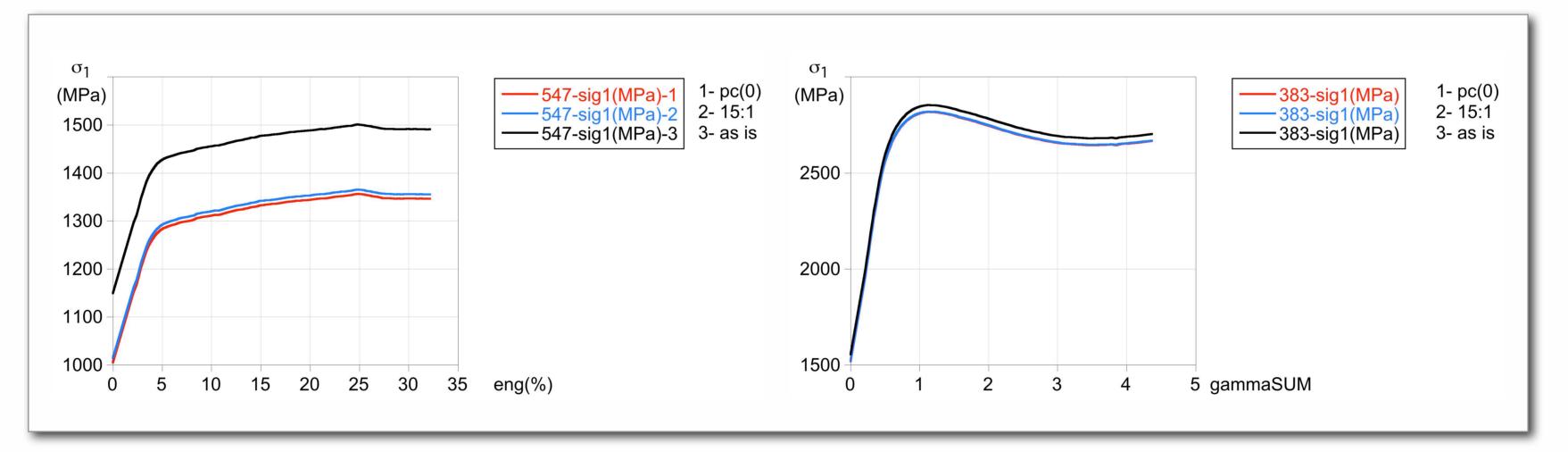


1532



#### effect of choices

 $\sigma_3 : \sigma_1 = 15:1$ 



### options E

I: sigI(0) = sig3(0) = pc(0)2: sig I(0) = sig 3(0) =1/16\*F(0)/A(0)+15/16\*pc(0)3: sig I(0)=F(0)/A(0) and sig 3(0)=pc(0)

The measured confining pressure pc(0) at the start of the experiment and the value of  $\sigma_1(0)$ seldom coincide.

 $\sigma_1(0)$  = load at start divided by cross sectional area of the loading piston = F(0)/A0. Generally,  $\sigma_1(0)$  is not evaluated, and the assumption is that  $\sigma_1(0) = \sigma_3(0) = pc(0)$ .

However, since the pressure inside the vessel is affected through the  $\sigma_{I}$ - and the  $\sigma_{I}$ -piston, and assuming that any differences between the load on these to pistons evens out, the 'average pressure' inside the vessel can be figured out. It depends on the relative cross sectional areas of the pistons through which the pressure (actually the load) is applied. The diameter of the  $\sigma_1$ -piston = 1/4", the outer diameter of the  $\sigma_3$ -piston = I", the ratio is 1:15.



## summary of options – corrections and calculations

### option

- stiffness correction subtracts the elastic distortion of the apparatus ( $\Delta$ mm) from d as a function of F: typical value for 'distortion'  $\approx 10 \, \mu m/kN$ dc = d - 'distortion'(F)
- definition of the hitpoint
- friction correction subtracts some force ( $\Delta F$ ) from the F as a function of d: Fc = F - friction'(d)typical value for 'friction' ≈ 1000 N/mm
- salt correction adds confining pressure ( $\Delta$ MPa) to medium inside vessel as a function of d, Pc and T: Pc = Pc + 'slope'typical value for 'slope'  $\approx 35$  MPa/mm for NaCl, 17 MPa for KI at Pc=IGPa, T=600°C, increasing with Pc, decreasing with T
- area correction (not really a correction) calculates the (non-linear) relative change of A as a function of dc: typically:  $A(dc)/A_0 = L_0 / (L_0 - L)$ axial experiments: sample cross section:  $A(dc)/A_0 > 1$ typically:  $A(dc)/A_0 = type ACF, cos, cos^2$  $A(dc)/A_0 < I$ shear experiments: piston overlap:
- definition of the starting values for  $\sigma_1$  and  $\sigma_3$
- = axial displacement of  $\Delta \sigma$  piston
- dc = displacement of piston inside vessel 'experienced by sample' (shortening of sample)
- F = applied load
- Fc = load applied to sample inside vessel 'felt by sample' (loading of sample)
- Pc = confining pressure
- = temperature
- = cross sectional area of sample (axial) of piston overlap (shear)
- = length of sample

 $\sigma_1$  piston should be called delta  $\Delta\sigma$  piston or load piston σ<sub>3</sub> piston should be called confining pressure piston

### effect of options

